

EECS 144/244: System Modeling, Analysis, and Optimization

Continuous Systems

Lecture: Linear Systems

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April 30, 2013

1 Systems with Inputs

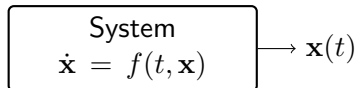
2 Linear Systems

3 Controllability

4 Transfer Function

Autonomous Systems

So far we worked mostly on systems of the form

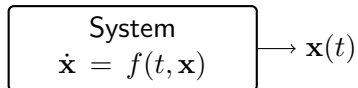


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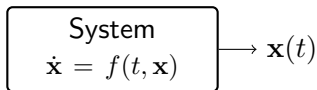
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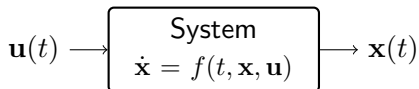
Given \mathbf{x}_0 , it produces a unique solution, called *flow* $\mathbf{x}(t, \mathbf{x}_0)$

Systems with Inputs

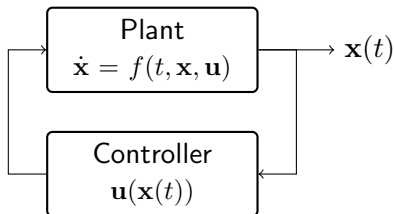
Closed system



Open loop



Closed-loop



Systems with Inputs

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OK if $F(t, \mathbf{x}(t)) = f(t, \mathbf{x}(t), \mathbf{u}(t))$ satisfies the same conditions as before (continuous, (locally) Lipschitz)

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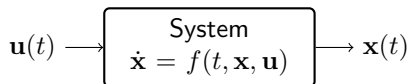
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OK if $F(t, \mathbf{x}(t)) = f(t, \mathbf{x}(t), \mathbf{u}(t))$ satisfies the same conditions as before (continuous, (locally) Lipschitz)

Although, the story is more complex

- ▶ Typically inputs are discontinuous (step functions)
- ▶ Complex feedback can create algebraic loops resulting in DAEs rather than ODEs (consult David)

Some Properties of Interest



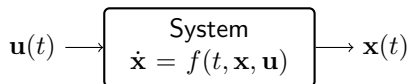
Stability

For any “reasonable” input \mathbf{u} , \mathbf{x} does not “explode” in some sense.

Different notions:

- ▶ Bounded Input Bounded Output: $\forall t, \|\mathbf{u}(t)\| < C_u \Rightarrow \forall t, \|\mathbf{x}(t)\| \leq C_x$
- ▶ Globally asymptotic: $\forall \mathbf{x}(0), \lim_{t \rightarrow \infty} \mathbf{x}(t) = 0$
- ▶ etc

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For linear systems, these questions are well understood

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Linear Functions

A *linear* function is a mapping which satisfies :

1. Additivity: $f(x + y) = f(x) + f(y)$
2. Scaling: $f(\lambda x) = \lambda f(x)$

or equivalently,

$$\forall \alpha, \beta, f(\alpha x + \beta y) = \alpha f(x) + \beta f(y)$$

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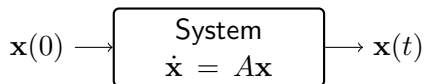
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Linear functions from \mathbb{R}^n to \mathbb{R}^m ? $f(\mathbf{x}) = A\mathbf{x}$ where A is an $m \times n$ matrix.

Linear Systems

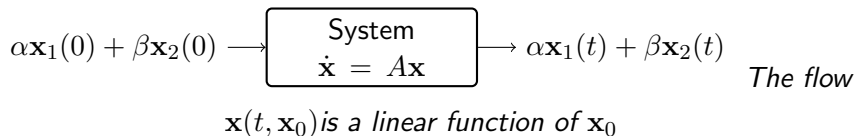
Linear dynamical systems



The flow $\mathbf{x}(t, \mathbf{x}_0)$ is a linear function of \mathbf{x}_0

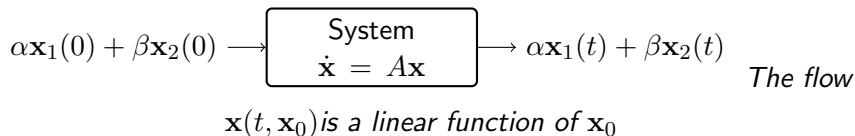
Linear Systems

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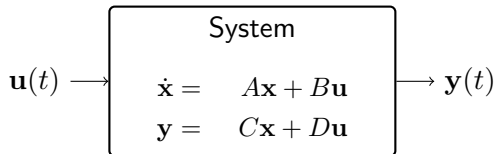


Linear Systems

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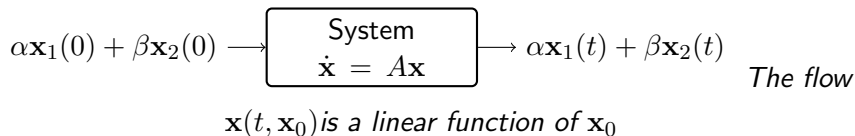
Linear control systems



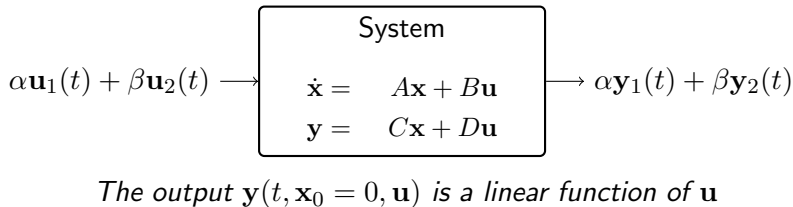
The output $\mathbf{y}(t, \mathbf{x}_0 = 0, \mathbf{u})$ is a linear function of \mathbf{u}

Linear Systems

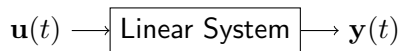
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Principle of Superposition



Principle of Superposition

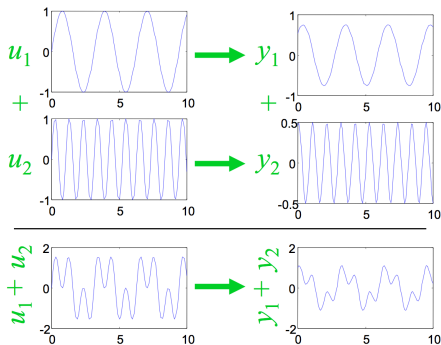
$$\alpha \mathbf{u}_1(t) + \beta \mathbf{u}_2(t) \longrightarrow \boxed{\text{Linear System}} \longrightarrow \alpha \mathbf{y}_1(t) + \beta \mathbf{y}_2(t)$$

Allows to characterize infer sets of behaviors from typical simple inputs

- ▶ Step signals $u(t) = 1(t)$
- ▶ Sinusoidal signals $u(t) = A \sin(\omega t)$

Principle of Superposition

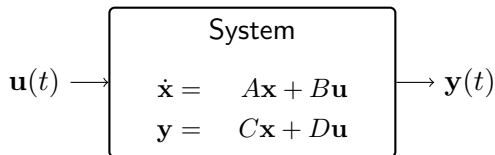
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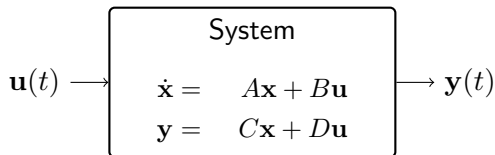
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Remarks



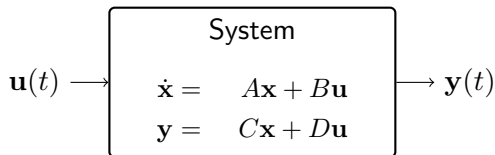
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- ▶ If $\mathbf{x}_0 \neq 0$, the system is not linear, strictly speaking. However, it is *affine*

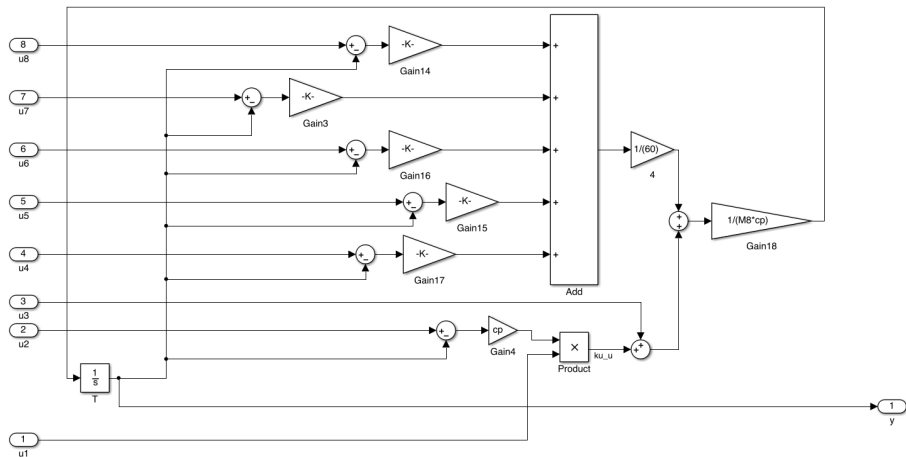
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- ▶ If $\mathbf{x}_0 \neq 0$, the system is not linear, strictly speaking. However, it is *affine*
- ▶ So is $f(\mathbf{x}, \mathbf{u}) = A\mathbf{x} + B\mathbf{u}$, by the way.

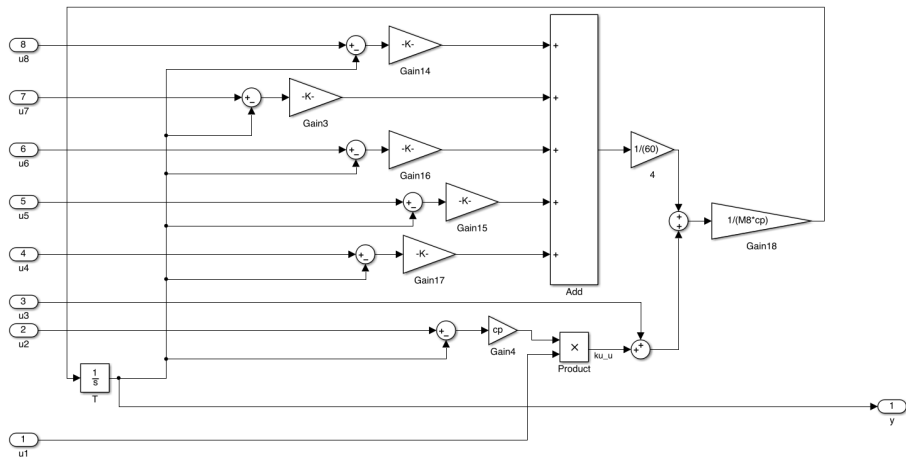
Example

Is this system linear ?



Example

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Solutions of Linear Systems

Recall that $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x}$, $\mathbf{x}(t=0) = \mathbf{x}_0$

admits the solution $\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}_0$

Where $e^{\mathbf{A}t}$ is matrix exponentiation

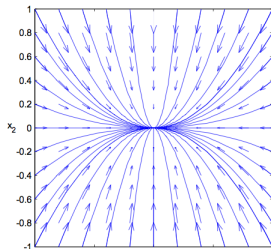
$$e^{t\mathbf{A}} = \mathbf{I}_n + \mathbf{A}t + \frac{t^2}{2!}\mathbf{A}^2 + \frac{t^3}{3!}\mathbf{A}^3 + \dots$$

The stability depends on the Eigenvalues of A

Eigenstructure of Linear Systems

Real e-values

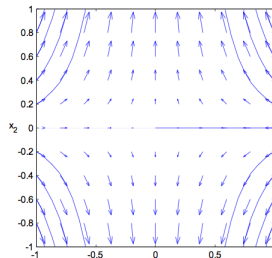
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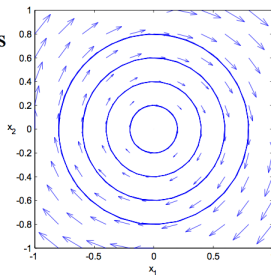
$$\operatorname{Re}(\lambda_i) < 0$$

$$\operatorname{Re}(\lambda_j) > 0$$



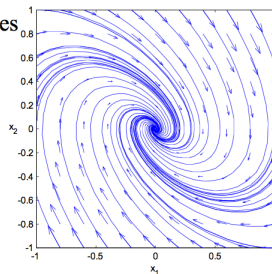
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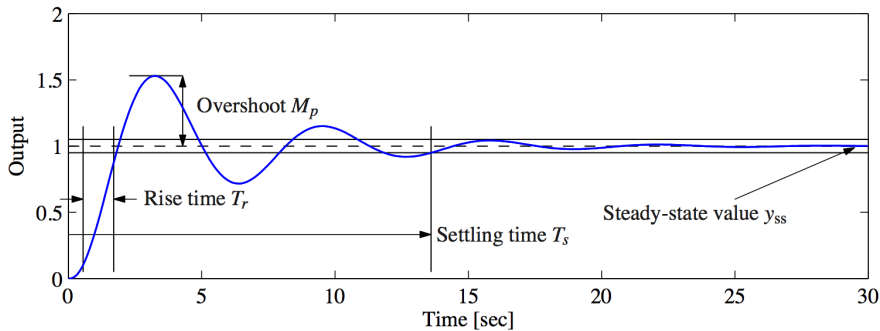
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Step response for LTI systems

$$u(t) = \begin{cases} 0 & t = 0 \\ 1 & t > 0 \end{cases}$$

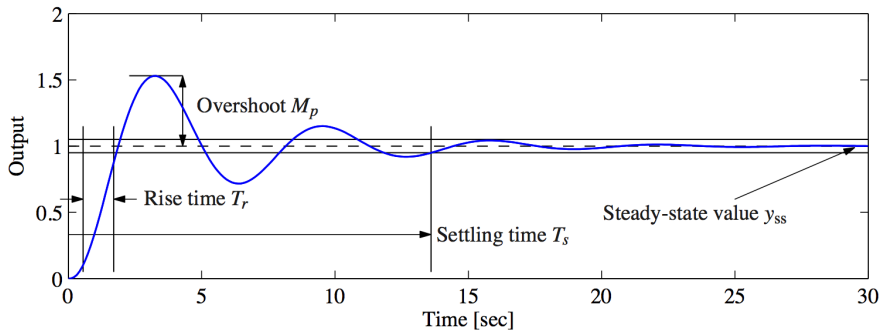
Equivalent to $\dot{\mathbf{z}} = \mathbf{A}\mathbf{z}$ with $\mathbf{z} = \mathbf{x} + \mathbf{A}^{-1}\mathbf{B}$



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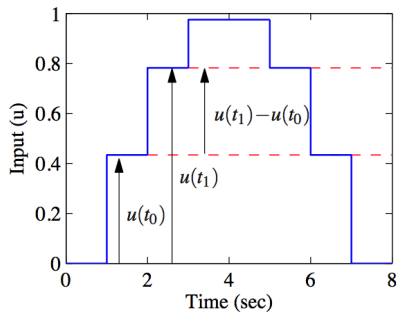
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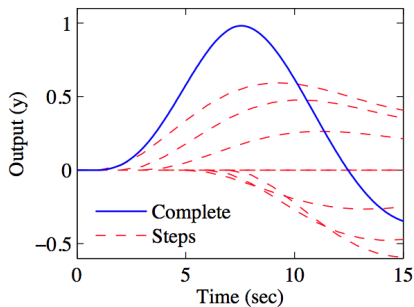


Piecewise Constant Inputs

Superposition of step responses



(a) Piecewise constant input



(b) Output response

LTI Systems Solution

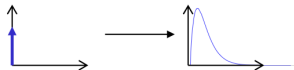
$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

$$\longrightarrow y(t) = \underbrace{Ce^{At}x(0)}_{\text{homogeneous}} + ???$$

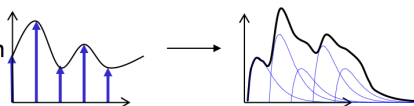
Impulse response, $h(t) = Ce^{At}B$

- Response to input “impulse”
- Equivalent to “Green’s function”



Linearity \Rightarrow compose response to arbitrary $u(t)$ using *convolution*

- Decompose input into “sum” of shifted impulse functions
- Compute impulse response for each
- “Sum” impulse response to find $y(t)$



Complete solution: use integral instead of “sum”

$$y(t) = Ce^{At}x(0) + \int_{\tau=0}^t Ce^{A(t-\tau)}Bu(\tau)d\tau + Du(t)$$

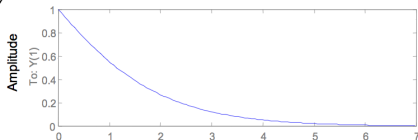
- linear with respect to initial condition *and* input
- 2X input \Rightarrow 2X output when $x(0) = 0$

Matlab Tools for Linear Systems

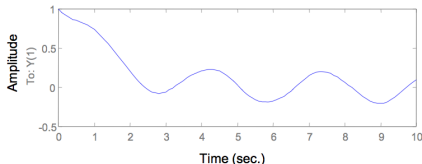
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```
A = [-1 1; 0 -1]; B = [0; 1];  
C = [1 0]; D = [0];  
x0 = [1; 0.5];  
  
sys = ss(A,B,C,D);  
initial(sys, x0);  
impulse(sys);  
  
t = 0:0.1:10;  
u = 0.2*sin(5*t) + cos(2*t);  
lsim(sys, u, t, x0);
```

Initial Condition Results



Linear Simulation Results



Other MATLAB commands

- gensig, square, sawtooth – produce signals of diff. types
- step, impulse, initial, lsim – time domain analysis
- bode, freqresp, evalfr – frequency domain analysis

ltiview – linear
time invariant
system plots

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Controllability


Consider the SISO LTI system:

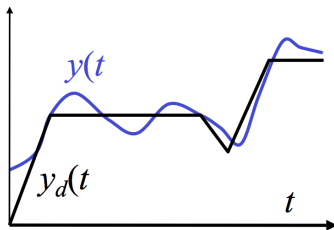
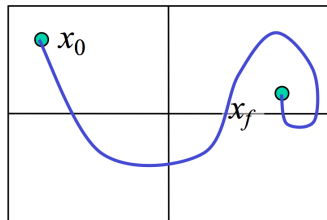
$$\begin{aligned}\dot{\mathbf{x}} &= A\mathbf{x} + Bu & \mathbf{x} \in \mathcal{R}^n, u \in \mathcal{R} \\ y &= C\mathbf{x} + Du & y \in \mathcal{R}\end{aligned}$$

is controllable if there exists $u(t)$ such that $\mathbf{x}(0) = \mathbf{x}_0$ and $\mathbf{x}(T) = \mathbf{x}$.

Note



- ▶ In , controllability is referred to as *reachability*
- ▶ Controllability enables the more general problem of tracking



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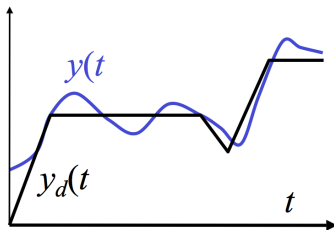
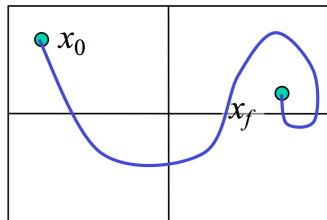
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Controllability Matrix

The controllability matrix for a linear system is given by:

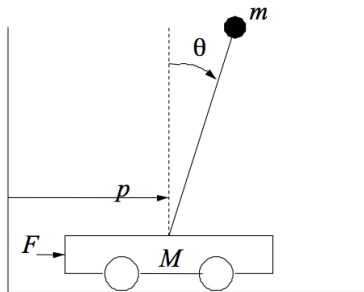
$$W_r = [B \quad AB \quad \dots \quad A^{n-1}B] .$$

A linear system is controllable if and only if the controllability matrix is *invertible*

Note

- ▶ In the general MIMO case, the condition becomes W_r has rank n
- ▶ Very simple test to do (e.g. in Matlab, concat matrices, run `rank`)

Example: Linearized pendulum on a cart



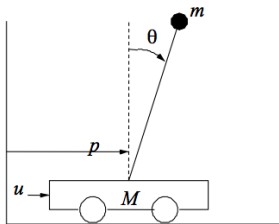
Question: can we locally control the position of the cart by proper choice of input?

Approach: look at the linearization around the upright position (good approximation to the full dynamics if θ remains small)

$$\frac{d}{dt} \begin{bmatrix} p \\ \theta \\ \dot{p} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & \frac{m^2 l^2 g}{M_t J_t - m^2 l^2} & \frac{-c J_t}{M_t J_t - m^2 l^2} & \frac{-\gamma J_t l m}{M_t J_t - m^2 l^2} \\ 0 & \frac{M_t m g l}{M_t J_t - m^2 l^2} & \frac{-c l m}{M_t J_t - m^2 l^2} & \frac{-\gamma M_t}{M_t J_t - m^2 l^2} \end{bmatrix} \begin{bmatrix} p \\ \theta \\ \dot{p} \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{J_t}{M_t J_t - m^2 l^2} \\ \frac{l m}{M_t J_t - m^2 l^2} \end{bmatrix} u$$
$$y = \begin{bmatrix} 1 & 0 & 0 & 0 \end{bmatrix} x,$$

Note here u is F .

Example: Linearized pendulum on a cart (cont'd)



$$\frac{d}{dt} \begin{bmatrix} p \\ \theta \\ \dot{p} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & \frac{m^2 l^2 g}{\mu} & \cancel{\frac{-c J_t}{\mu}} & \cancel{\frac{-\gamma J_t l m}{\mu}} \\ 0 & \frac{M_t m g l}{\mu} & \cancel{\frac{-c l m}{\mu}} & \cancel{\frac{-\gamma M_t}{\mu}} \end{bmatrix} \begin{bmatrix} p \\ \theta \\ \dot{p} \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{J_t}{\mu} \\ \frac{l m}{\mu} \end{bmatrix} u$$

$\mu = M_t J_t - m^2 l^2$

• Simplify by setting $c, \gamma = 0$

Reachability matrix

$$W_r = \begin{bmatrix} \begin{bmatrix} 0 & \frac{J_t}{\mu} \\ 0 & \frac{l m}{\mu} \\ \frac{J_t}{\mu} & 0 \\ \frac{l m}{\mu} & 0 \end{bmatrix} & \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} & \begin{bmatrix} \frac{g l^3 m^3}{\mu^2} \\ \frac{g l^2 m^2 (m+M)}{\mu^2} \\ 0 \\ 0 \end{bmatrix} \\ B & AB & A^2 B & A^3 B \end{bmatrix}$$

- Full rank as long as constants are such that columns 1 and 3 are not multiples of each other

- \Rightarrow reachable as long as

$$\det(W_r) = \frac{g^2 l^4 m^4}{\mu^4} \neq 0$$

- \Rightarrow can “steer” linearization between points by proper choice of input

State Feedback

A state feedback law has the form $\mathbf{u} = -Kx + k_r r$ where r is the reference value for the output.

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The closed loop dynamics for the system are given by

$$\dot{\mathbf{x}} = (A - BK)\mathbf{x} + Bk_r r.$$

State Feedback

A state feedback law has the form $\mathbf{u} = -K\mathbf{x} + k_r r$ where r is the reference value for the output.

The closed loop dynamics for the system are given by

$$\dot{\mathbf{x}} = (A - BK)\mathbf{x} + Bk_r r.$$

The stability of the system is determined by the stability of the matrix $A - BK$. The equilibrium point and steady state output are given by

$$\mathbf{x}_e = -(A - BK)^{-1} Bk_r r \quad \mathbf{y}_e = C\mathbf{x}_e.$$

Choosing k_r as $k_r = -1/(C(A - BK)^{-1}B)$ gives $\mathbf{y}_e = r$.

Controllability

If a system is controllable, then there exists a feedback law of the form

$$u = -Kx + k_r r$$

the gives a closed loop system with an arbitrary characteristic polynomial.

Hence the eigenvalues of a controllable linear system can be placed arbitrarily through the use of an appropriate feedback control law.

Limits of poles placement

Saturation of inputs, transient behaviors, etc.

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Popular state feedback control laws in practice:

PID controller

$$u(t) = k_p e(t) + k_i \int_0^t e(\tau) d\tau + k_d \frac{de}{dt},$$

where $e(t)$ is the difference between $y(t)$ and the reference value $r(t)$.

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Linear quadratic regulator

Minimizes

$$\tilde{J} = \int_0^\infty (\mathbf{x}^T Q_x \mathbf{x} + \mathbf{u}^T Q_u \mathbf{u}) dt.$$

for (user)-given matrices Q_x and Q_u .

1 Systems with Inputs

2 Linear Systems

3 Controllability

4 Transfer Function

Laplace Transform

The Laplace Transform of a function $f : \mathbb{R}^+ \rightarrow \mathbb{R}$ is the function $F = \mathcal{L}\{f\} : \mathbb{C} \rightarrow \mathbb{C}$ such that

$$\forall s \in \mathbb{C}, F(s) = \mathcal{L}\{f\}(s) = \int_0^{\infty} e^{-st} f(t) dt$$

Main properties:

- ▶ **Linear:** $\mathcal{L}\{af + bg\} = a\mathcal{L}\{f\} + b\mathcal{L}\{g\}$
- ▶ **Derivative:** $\mathcal{L}\left\{\frac{df}{dt}\right\}(s) = s \cdot \mathcal{L}\{f\} - f(0)$

In particular, if $f(0) = 0$, we get $\boxed{\mathcal{L}\left\{\frac{df}{dt}\right\} = sF(s)}$

Solving differential equations with Laplace Transform

$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \cdots + a_n y = b_0 \frac{d^m u}{dt^m} + b_1 \frac{d^{m-1} u}{dt^{m-1}} + \cdots + b_m u$$

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Assuming that initial conditions are 0 and applying the Laplace transform, we get:

$$(s^n + a_1 s^{n-1} + \cdots + a_n)Y(s) = (b_0 s^m + b_1 s^{m-1} + \cdots + b_m)U(s)$$

We note $d(s) = (s^n + a_1 s^{n-1} + \cdots + a_n)$ and $n(s) = (b_0 s^m + b_1 s^{m-1} + \cdots + b_m)$.

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$$Y(s) = \frac{n(s)}{d(s)}U(s) = H(s)U(s)$$

$H(s)$ is called the transfer function of the system

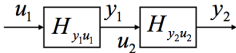
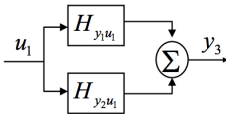
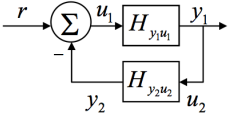
Transfer Functions and State space representation

More generally, the transfer function for a linear system (A, B, C, D) is given by

$$H(s) = \frac{n(s)}{d(s)} = C(sI - A)^{-1}B + D, \quad s \in \mathbb{C}$$

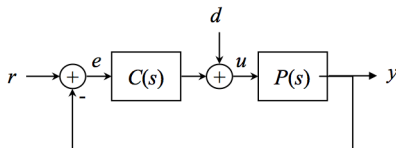
- ▶ the roots of $d(s)$ are called the **poles of the system** and correspond to the eigenvalues of the dynamics matrix A
- ▶ the roots of the polynomial $b(s)$ are called the zeros of the system and correspond to exponential signals whose transmission is blocked by the system.

Transfer functions and block diagram Algebra

Type	Diagram	Transfer function
Series		$H_{y_2 u_1} = H_{y_2 u_2} H_{y_1 u_1} = \frac{n_1 n_2}{d_1 d_2}$
Parallel		$H_{y_3 u_1} = H_{y_2 u_1} + H_{y_1 u_1} = \frac{n_1 d_2 + n_2 d_1}{d_1 d_2}$
Feedback		$H_{y_1 r} = \frac{H_{y_1 u_1}}{1 + H_{y_1 u_1} H_{y_2 u_2}} = \frac{n_1 d_2}{n_1 n_2 + d_1 d_2}$

Transfer functions and block diagram Algebra

Basic idea: treat transfer functions as multiplication, write down equations



$$y = P(s)u$$

$$u = d + C(s)e$$

$$e = r - y$$

Manipulate equations to compute desired signals

$$e = r - y$$

$$= r - P(s)u$$

$$= r - P(s)(d + C(s)e)$$

$$(1 + P(s)C(s))e = r - P(s)d$$

$$e = \underbrace{\frac{1}{1 + P(s)C(s)}}_{H_{er}} r - \underbrace{\frac{P(s)}{1 + P(s)C(s)}}_{H_{ed}} d$$

Note: linearity
gives super-
position of terms

Usual Transfer Functions

Type	ODE	Transfer Function
Integrator	$\dot{y} = u$	$\frac{1}{s}$
Differentiator	$y = \dot{u}$	s
First-order system	$\dot{y} + ay = u$	$\frac{1}{s + a}$
Double integrator	$\ddot{y} = u$	$\frac{1}{s^2}$
Damped oscillator	$\ddot{y} + 2\zeta\omega_0\dot{y} + \omega_0^2y = u$	$\frac{1}{s^2 + 2\zeta\omega_0s + \omega_0^2}$
PID controller	$y = k_p u + k_d \dot{u} + k_i \int u$	$k_p + k_d s + \frac{k_i}{s}$
Time delay	$y(t) = u(t - \tau)$	$e^{-\tau s}$

References



Feedback Systems: An introduction for scientists and engineers

Karl J. Astrom and Richard M. Murray

http://www.cds.caltech.edu/~murray/amwiki/index.php/Main_Page